AMENDMENTS TO THE SPECIFICATION

Please replace paragraph [0011] with the following amended paragraph:

[0011] Fig. 6 shows in solid lines two rocker pressure members 14_1 and 14_2 of an extended plate-link chain (not shown). Both rolling surfaces 18_1 and 18_2 come into contact with each other at point B1. The inner ~~side~~ surface of a link ~~directed~~ opening that faces toward the left in ~~accordance with~~ as viewed in Fig. 6 is supported on the surface of the rocker pressure member 14_2 that lies opposite to the rolling surface 18_2. [[A]] The inner surface of a link ~~directed to~~ opening that faces toward the right as viewed in Fig. 6 is supported on the surface of rocker pressure member 14_1 that lies opposite to surface 18_1, as is clear from Fig. 3. When both links are tilted relative to each other, they take the rocker pressure members 14_1 and 14_2 along with them, so that their opposed rolling surfaces 18_1 and 18_2 roll against each other, and the point of contact B1 (or the contact line) moves toward contact point B2. The tilted state of the rocker pressure members is shown in dashed lines, whereby their individual tilt relative to the initial position is $\alpha/2$ so that the total tilt angle (Fig. 4) is α. As is apparent from Fig. 6, the centers M1 and M2 of the rocker pressure ~~members~~ member cross sections move apart from each other upon tilting. Collectively, in that way an effective link elongation results as a function of tilt angle α, as it is represented in Fig. 7. The elongation begins in the illustrated example at a tilt angle from 0 to 3°, since when the plate-link chain is extended the rocker pressure members are situated in an orientation relative to each other in which they touch outside the

longitudinal mid-plane of the plate-link chain. In that way greater bend angles are possible.

Please replace paragraph [0030] with the following amended paragraph:

[0030] A first solution of the object of the invention is achieved with a plate-link chain that is composed of links and rocker pressure member pairs that extend transversely through the plate-link chain, which are arranged in several rows one after the other relative to the transverse direction of the plate-link chain, whereby each link is penetrated by two rocker pressure member pairs, one following the other in the longitudinal direction of the plate-link chain. Each rocker pressure member pair penetrates at least two links of different rows offset relative to each other in the longitudinal direction of the plate-link chain. Surfaces of rocker pressure member pairs facing away from each other in the longitudinal direction of the plate-link chain are in contact with opposite end sides of openings of links that are offset relative to each other. Surfaces of the rocker pressure members of a rocker pressure member pair facing each other form rolling surfaces on which the rocker pressure members roll on each other when the plate-link chain is curved, and lateral end faces of the rocker pressure member pairs are formed for contact on conical surfaces of the conical disk pairs. The plate-link chain is characterized in that the rolling surfaces of the rocker pressure members are formed as freeform surfaces in such a way that changes in the spacing between centers of cross sections of rocker pressure members rolling on

one another during a mutual tilting of successive links in the longitudinal direction of the plate-link chain are at least partially compensated.

Please replace paragraph [0031] with the following amended paragraph:

[0031] A further solution of the object of the invention is achieved with a plate-link chain that is composed of links and rocker pressure member pairs that extend transversely through the plate-link chain, which are arranged in several rows one after the other relative to the transverse direction of the plate-link chain, whereby each link is penetrated by two rocker pressure member pairs, one following the other in the longitudinal direction of the plate-link chain. Each rocker pressure member pair penetrates at least two links of different rows offset relative to each other in the longitudinal direction of the plate-link chain, surfaces of rocker pressure member pairs facing away from each other in the longitudinal direction of the plate-link chain are in contact with opposite end sides of openings of links that are offset relative to each other, surfaces of the rocker pressure members of a rocker pressure member pair facing each other form rolling surfaces on which the rocker pressure members roll on each other when the plate-link chain is curved, and lateral end faces of the rocker pressure member pairs are formed for contact on conical surfaces of the conical disk pairs, and the links of at least one of the adjacently arranged rows have different lengths so that the distance between the end faces of the rocker pressure members is different, which plate-link chain is characterized in that the rolling surfaces of the rocker pressure members are constructed as freeform surfaces, that the influence of the

length of the ~~rocker members~~ links on the shortening of the effective chain length during rotation about a circular arc (polygon effect) is at least partially compensated.

Please replace paragraph [0040] with the following amended paragraph:

[0040] The rolling surfaces of the rocker pressure members formed in accordance with the invention are not formed as segments of a circular cylinder, but as freeform surfaces, which are formed corresponding to the conditions of the plate-link chain (link length; minimal and maximal radii (~~Fig. 4~~ see Figs. 4A and 4B); link connection; stresses).

Please replace paragraph [0041] with the following amended paragraph:

[0041] In ~~Fig~~ Fig. 5 an example of a rolling surface formed in accordance with the invention is indicated by dashed lines, whereby the cross section of the rocker pressure members in that example is equal over their entire length. As is apparent, the distance that a point P (or a line) has from the point O (or a line), which is the center of curvature of the radius of curvature R_0 that the rolling surface 18 has at point P_0, changes with the angle β. In the illustrated example, the distance R becomes smaller with an increasing angle β, so that during rolling of the rocker pressure member pairs 14, the increase in distance between the centers of the rocker pressure members that roll on each other is reduced. According to the requirements, the distance R can also increase with increasing β, or first increase and then decrease, or the opposite.